

**University of Stuttgart** Institute for Nonlinear Mechanics Advances in ELAstoDYNamics of architeted materials and BIOmaterials

Finite element formulations for constrained spatial nonlinear beam theories<sup>\*</sup>

\*submitted to Math. Mech. Solids

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# Goal

single finite element formulation for dynamic simulations of classical nonlinear beams

- shear-deformable, shear-rigid and inextensible shear-rigid beams
- large displacements and finite strains
- hyperelastic constitutive laws
- precurved reference configurations



### Idea

# beam theories as constrained theories in which

### constraints can be switched on and off

### Kinematics of shear-deformable beam



	reference configuration	current configuration
centerline	$\mathbf{r}_0(\xi)$	${f r}(\xi,t)$
directors	$\mathbf{D}_i(\xi) = \mathbf{R}_0(\xi)\mathbf{e}_i$	$\mathbf{d}_i(\xi, t) = \mathbf{R}(\xi, t)\mathbf{e}_i$

reference arc-length

$$s = \int_0^{\xi} \frac{\|\mathbf{r}_0'(\bar{\xi})\| \mathrm{d}\bar{\xi}}{=: J(\bar{\xi})}$$

$$\frac{\partial f}{\partial s}(\xi,t) := \frac{\partial f}{\partial \xi}(\xi,t) \frac{1}{J(\xi)} = \frac{f'(\xi,t)}{J(\xi)}$$

## **Objective strain measures**



- $\Gamma_{i} = \frac{1}{J} \mathbf{r}' \cdot \mathbf{d}_{i}$  $\mathbf{\Gamma} = \Gamma_{i} \mathbf{D}_{i} = \mathbf{\Lambda}^{\mathrm{T}} \frac{\partial \mathbf{r}}{\partial s}$
- $\Gamma_1$  dilatation
- $\Gamma_2, \Gamma_3$  shear
- $\lambda = \| \Gamma \|\,$  stretch

$$K_i = \frac{1}{2J} \varepsilon_{ijk} \mathbf{d}_k \cdot \mathbf{d}'_j, \ K_i^0 = \frac{1}{2J} \varepsilon_{ijk} \mathbf{D}_k \cdot \mathbf{D}'_j$$

$$\mathbf{K} - \mathbf{K}_0 = (K_i - K_i^0)\mathbf{D}_i = \operatorname{ax}\left(\mathbf{\Lambda}^{\mathrm{T}}\frac{\partial\mathbf{\Lambda}}{\partial s}\right)$$

- 
$$K_1 - K_1^0$$
 torsion  
-  $K_2 - K_2^0, K_3 - K_3^0$  flexure

### Internal virtual work

total strain energy stored in the beam

$$E = \int_{I} U(\xi, t) J(\xi) d\xi = \int_{I} W(\Gamma_{i}(\xi, t), K_{i}(\xi, t); \xi) J(\xi) d\xi$$
$$W(\Gamma_{i}, K_{i}; \xi) = \frac{1}{2} k_{t} (K_{1} - K_{1}^{0})^{2} + \frac{1}{2} k_{b_{2}} (K_{2} - K_{2}^{0})^{2} + \frac{1}{2} k_{b_{3}} (K_{3} - K_{3}^{0})^{2} + \frac{1}{2} k_{e} (\lambda(\Gamma_{i}) - 1)^{2} + \frac{1}{2} k_{s_{2}} \Gamma_{2}^{2} + \frac{1}{2} k_{s_{3}} \Gamma_{3}^{2}$$

internal virtual work

$$\delta W^{\text{int}} = -\delta E = -\int_{I} \left\{ \frac{\partial W}{\partial \Gamma_{i}} \delta \Gamma_{i} + \frac{\partial W}{\partial K_{i}} \delta K_{i} \right\} J d\xi$$
$$= -\int_{I} \left\{ \delta \mathbf{r}' \cdot n_{i} \mathbf{d}_{i} + \delta \mathbf{d}_{i} \cdot (n_{i} \mathbf{r}' + \varepsilon_{kji} \frac{m_{k}}{2} \mathbf{d}_{j}') + \delta \mathbf{d}_{j}' \cdot \varepsilon_{ijk} \frac{m_{i}}{2} \mathbf{d}_{k} \right\} d\xi$$

variation of strain measures

$$\delta\Gamma_i = \frac{1}{J} (\delta \mathbf{r}' \cdot \mathbf{d}_i + \delta \mathbf{d}_i \cdot \mathbf{r}') , \quad \delta K_i = \frac{1}{2J} \varepsilon_{ijk} (\delta \mathbf{d}_k \cdot \mathbf{d}'_j + \delta \mathbf{d}'_j \cdot \mathbf{d}_k)$$

### Internal virtual work

total strain energy stored in the beam

$$E = \int_{I} U(\xi, t) J(\xi) d\xi = \int_{I} W(\Gamma_{i}(\xi, t), K_{i}(\xi, t); \xi) J(\xi) d\xi$$
$$W(\Gamma_{i}, K_{i}; \xi) = \frac{1}{2} k_{t} (K_{1} - K_{1}^{0})^{2} + \frac{1}{2} k_{b_{2}} (K_{2} - K_{2}^{0})^{2} + \frac{1}{2} k_{b_{3}} (K_{3} - K_{3}^{0})^{2} + \frac{1}{2} k_{e} (\lambda(\Gamma_{i}) - 1)^{2} + \frac{1}{2} k_{s_{2}} \Gamma_{2}^{2} + \frac{1}{2} k_{s_{3}} \Gamma_{3}^{2}$$

internal virtual work

$$\delta W^{\text{int}} = -\delta E = -\int_{I} \left\{ \frac{\partial W}{\partial \Gamma_{i}} \delta \Gamma_{i} + \frac{\partial W}{\partial K_{i}} \delta K_{i} \right\} J d\xi$$
$$= -\int_{I} \left\{ \delta \mathbf{r}' \cdot n_{i} \mathbf{d}_{i} + \delta \mathbf{d}_{i} \cdot (n_{i} \mathbf{r}' + \varepsilon_{kji} \frac{m_{k}}{2} \mathbf{d}_{j}') + \delta \mathbf{d}_{j}' \cdot \varepsilon_{ijk} \frac{m_{i}}{2} \mathbf{d}_{k} \right\} d\xi$$

variation of strain measures

$$\delta\Gamma_i = \frac{1}{J} (\delta \mathbf{r}' \cdot \mathbf{d}_i + \delta \mathbf{d}_i \cdot \mathbf{r}') , \quad \delta K_i = \frac{1}{2J} \varepsilon_{ijk} (\delta \mathbf{d}_k \cdot \mathbf{d}'_j + \delta \mathbf{d}'_j \cdot \mathbf{d}_k)$$

unconstrained variation of directors

# **Constraint virtual work**



 $\mathbf{d}_i \otimes \mathbf{D}_i \not\in \mathrm{Orth}^+$ 

$$g_7 = \Gamma_2 = \mathbf{d}_2 \cdot \mathbf{r}' = 0$$
  

$$g_8 = \Gamma_3 = \mathbf{d}_3 \cdot \mathbf{r}' = 0$$
  

$$g_9 = \Gamma_1 = \mathbf{d}_1 \cdot \mathbf{r}' - 1 = 0$$

shear-rigidity & inextensibility

$$\delta W^{c} = \sum_{k=1}^{n_{c}} \int_{I} g_{k} \delta \lambda_{k} \, \mathrm{d}\xi + \int_{I} \delta g_{k} \lambda_{k} \, \mathrm{d}\xi$$

$$g_1 = \mathbf{d}_1 \cdot \mathbf{d}_1 - 1 = 0, \ g_4 = \mathbf{d}_1 \cdot \mathbf{d}_2 = 0$$
  

$$g_2 = \mathbf{d}_2 \cdot \mathbf{d}_2 - 1 = 0, \ g_5 = \mathbf{d}_1 \cdot \mathbf{d}_3 = 0$$
  

$$g_3 = \mathbf{d}_3 \cdot \mathbf{d}_3 - 1 = 0, \ g_6 = \mathbf{d}_2 \cdot \mathbf{d}_3 = 0$$

orthonormality of directors

$$\delta W^{\rm c} = \sum_{k=1}^{6} \int_{I} g_k \delta \lambda_k \,\mathrm{d}\xi + \int_{I} \delta g_k \lambda_k \,\mathrm{d}\xi$$

$$g_7 = \Gamma_2 = \mathbf{d}_2 \cdot \mathbf{r}' = 0$$
$$g_8 = \Gamma_3 = \mathbf{d}_3 \cdot \mathbf{r}' = 0$$

shear-rigidity

 $n_c = 6$  shear-deformable  $n_c = 8$  shear-rigid  $n_c = 9$  inextensible & shear-rigid

### **External virtual work**

$$\delta \mathbf{d}_i = \underline{\operatorname{ax}(\delta \mathbf{R} \mathbf{R}^{\mathrm{T}})}_{= \delta \boldsymbol{\phi}} \times \mathbf{d}_i \quad \Rightarrow \delta \boldsymbol{\phi} = \frac{1}{2} \mathbf{d}_i \times \delta \mathbf{d}_i \quad \text{virtual rotation}$$

$$\delta W^{\text{ext}} = \int_{I} \left\{ \delta \mathbf{r} \cdot \overline{\mathbf{n}} + \delta \phi \cdot \overline{\mathbf{m}} \right\} J \, \mathrm{d}\xi + \sum_{k=0}^{1} \left\{ \delta \mathbf{r} \cdot \overline{\mathbf{n}}_{k} + \delta \phi \cdot \overline{\mathbf{m}}_{k} \right\} \Big|_{\xi = \xi_{k}}$$

 $\overline{\mathbf{n}}, \overline{\mathbf{m}}$  line distributed forces and moments  $\overline{\mathbf{n}}_k, \overline{\mathbf{m}}_k$  boundary forces and moments

# Virtual work of inertia effects

$$\delta W^{\mathrm{dyn}} = -\int_{I} \left\{ \delta \mathbf{r} \cdot (A_{\rho_{0}} \ddot{\mathbf{r}} + B_{\rho_{0}}^{\alpha} \ddot{\mathbf{d}}_{\alpha}) + \delta \mathbf{d}_{\alpha} \cdot (B_{\rho_{0}}^{\alpha} \ddot{\mathbf{r}} + C_{\rho_{0}}^{\alpha\beta} \ddot{\mathbf{d}}_{\beta}) \right\} J \,\mathrm{d}\xi$$



no  $\ddot{\mathbf{d}}_1$  or  $\delta \mathbf{d}_1$  contribution!

#### reference density

$$A_{\rho_0}(\xi) = \int_{A(\xi)} \rho_0(\xi) \, \mathrm{d}A \, , \, B^{\alpha}_{\rho_0}(\xi) = \int_{A(\xi)} \rho_0(\xi) \theta_{\alpha} \, \mathrm{d}A \, , \, C^{\alpha\beta}_{\rho_0}(\xi) = \int_{A(\xi)} \rho_0(\xi) \theta_{\alpha} \theta_{\beta} \, \mathrm{d}A$$



# Principle of virtual work

$$\delta W = \delta W^{\text{int}} + \delta W^{\text{ext}} + \delta W^{\text{dyn}} + \delta W^{\text{c}} \stackrel{!}{=} 0 \quad \forall \delta \mathbf{r}, \delta \mathbf{d}_i, \delta \boldsymbol{\lambda}, t$$

#### discrete virtual work contributions

$$\begin{split} \delta W^{\text{int,h}} &= \delta \mathbf{q}^{\mathrm{T}} \mathbf{f}^{\text{int}}(\mathbf{q}) \\ \delta W^{\text{ext,h}} &= \delta \mathbf{q}^{\mathrm{T}} \mathbf{f}^{\text{ext}}(t,\mathbf{q}) \\ \delta W^{\text{dyn,h}} &= -\delta \mathbf{q}^{\mathrm{T}} \mathbf{M} \ddot{\mathbf{q}} \\ \delta W^{\text{c,h}} &= \delta \boldsymbol{\lambda}^{\mathrm{T}} \boldsymbol{g}(\mathbf{q}) + \delta \mathbf{q}^{\mathrm{T}} \mathbf{W}(\mathbf{q}) \boldsymbol{\lambda} \end{split}$$

internal generalized forces
external generalized forces
(singular) mass matrix
integrated constraints, constraint forces

equations of motion

$$\mathbf{M}\ddot{\mathbf{q}} - \mathbf{f}^{\text{int}}(\mathbf{q}) - \mathbf{f}^{\text{ext}}(t, \mathbf{q}) - \mathbf{W}(\mathbf{q})\boldsymbol{\lambda} = 0$$
$$\mathbf{g}(\mathbf{q}) = 0$$

differential algebraic equation of index 3, [1]

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statics 
$$\mathbf{R}(\mathbf{q}, \boldsymbol{\lambda}) = \begin{pmatrix} \mathbf{f}^{int}(\mathbf{q}) + \mathbf{f}^{ext}(\mathbf{q}) + \mathbf{W}(\mathbf{q})\boldsymbol{\lambda} \\ g(\mathbf{q}) \end{pmatrix} = 0$$

## Numerical validation - cantilever beam

first and second elliptic integral

$$F(\theta, p) = \int_0^{\theta} (1 - p^2 \sin^2 \tilde{\theta})^{-\frac{1}{2}} d\tilde{\theta} , \quad E(\theta, p) = \int_0^{\theta} (1 - p^2 \sin^2 \tilde{\theta})^{\frac{1}{2}} d\tilde{\theta}$$

replace external moment M by force Pwith rigid lever e = M/P

$$ks = F(\phi(s), p) - F(\phi_C, p) , \quad \cos \phi_C = ek/2p$$

$$k = \sqrt{P/k_{b_2}} \quad \text{solve for } p \quad \text{solve for } p$$

inextensibility:  $\sin \phi(L) = \sin \phi_B = 1/(p\sqrt{2})$ 

vertical and horizontal deflection

$$x(s) = 2p [\cos(\phi_B) - \cos(\phi)]/k$$
  

$$y(s) = -[2E(\phi_B, p) - 2E(\phi, p) + F(\phi, p) - F(\phi_B, p)]/k$$



-2

-4

6

 $I \times [2]$ 

4

 $\mathcal{X}$ 

## Numerical validation - helix

analytic solution (semi-inverse method)

$$\mathbf{r}^{*}(s) = R_{0} \sin \alpha(s) \mathbf{e}_{1} - R_{0} \cos \alpha(s) \mathbf{e}_{2} + cR_{0}\alpha(s) \mathbf{e}_{3}$$

$$\alpha(s) = 2\pi n s / L$$

$$c = h / (R_{0} 2\pi n)$$
height radius #coils



#### strain measures



### Numerical validation - objectivity

$$\Gamma_{i}^{h} = \frac{1}{J} \mathbf{r}^{h'} \cdot \mathbf{d}_{i}^{h} \qquad \mathbf{r}^{h,+} = \mathbf{Q}\mathbf{r}^{h} + \mathbf{c} \\ \mathbf{d}_{i}^{h,+} = \mathbf{Q}\mathbf{d}_{i}^{h} \qquad \Gamma_{i}^{h,+} = \frac{1}{J} (\mathbf{r}^{h,+})' \cdot \mathbf{d}_{i}^{h,+} = \Gamma_{i}^{h} \\ \mathbf{d}_{i}^{h,+} = \mathbf{Q}\mathbf{d}_{i}^{h} \qquad \mathbf{c} \in \mathbb{E}^{3}, \mathbf{Q} \in \text{Orth} \qquad K_{i}^{h,+} = \frac{1}{2J} \varepsilon_{ijk} \mathbf{d}_{k}^{h,+} \cdot (\mathbf{d}_{j}^{h,+})' = K_{i}^{h}$$





# Numerical validation - Wilberforce pendulum

#### pure vertical oscillation pure torsional oscillation MMMMMMM 400 [m] $\bigcirc$ -0.1200 3 $\aleph$ ()-0.2-200-400-2 $\mathbf{2}$ 6 8 10 121416 1820220 4

t

15

# Conclusion

- rather straight forward finite element formulation
- <u>shear-deformable</u>, <u>shear-rigid</u> and <u>inextensible shear-rigid</u> beams
- large displacements and finite strains
- hyperelastic constitutive laws
- precurved reference configurations
- discretization preserves objectivity of strain measures
- DAE solver is required
- locking for very slender beams
- orthonormality of directors only weakly satisfied
- computationally expensive